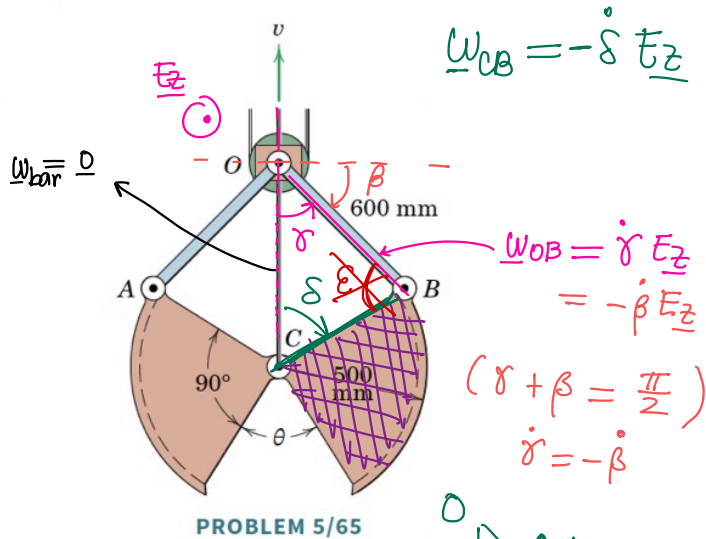
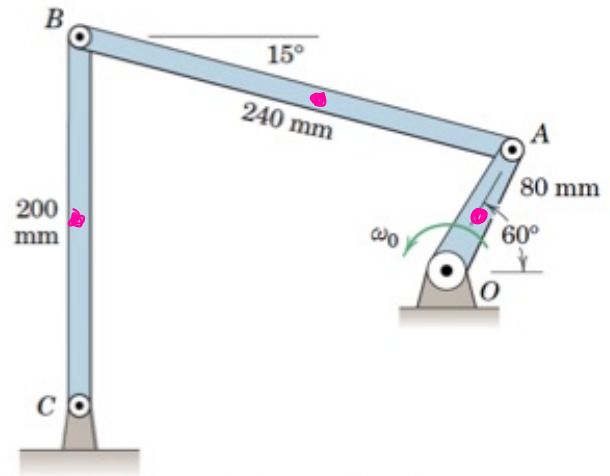


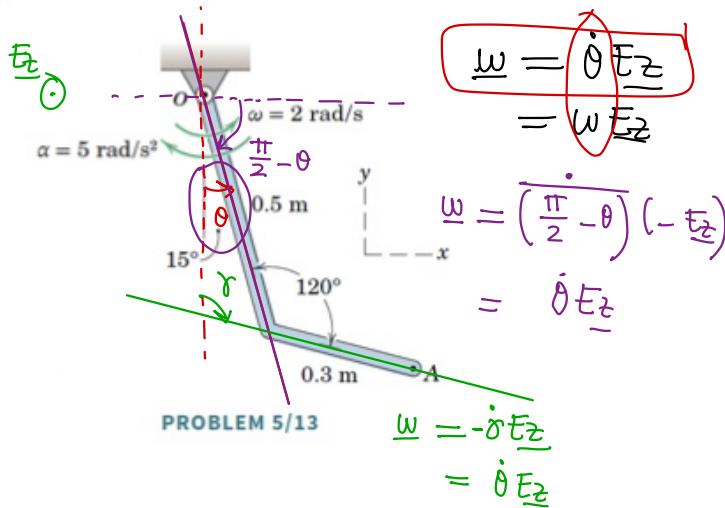
5/65 The elements of a simplified clam-shell bucket for a dredge are shown. The cable which opens and closes the bucket passes through the block at  $O$ . With  $O$  as a fixed point, determine the angular velocity  $\omega$  of the bucket jaws when  $\theta = 45^\circ$  as they are closing. The upward velocity of the control cable is 0.5 m/s as it passes through the block.



5/69 SS A four-bar linkage is shown in the figure (the ground "link"  $OC$  is considered the fourth bar). If the drive link  $OA$  has a counterclockwise angular velocity  $\omega_0 = 10$  rad/s, determine the angular velocities of links  $AB$  and  $BC$ .



5/13 The bent flat bar rotates about a fixed axis through point  $O$  with the instantaneous angular properties indicated in the figure. Determine the velocity and acceleration of point  $A$ .



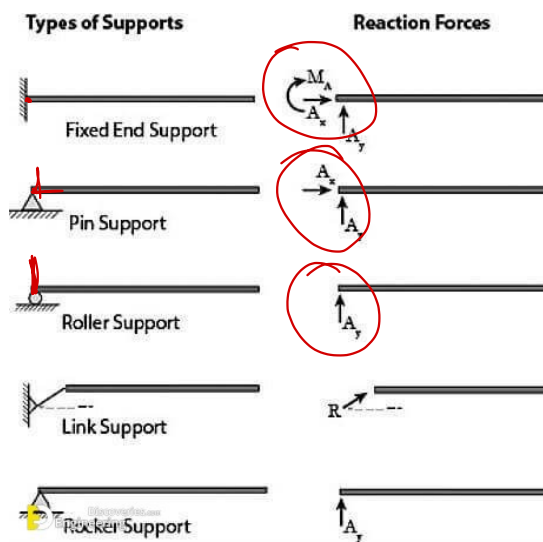
$$\frac{\sin \delta}{0.6} = \frac{\sin \gamma}{0.5}$$

$$\frac{\dot{\delta} \cos \delta}{0.6} = \frac{\dot{\gamma} \sin \gamma}{0.5}$$

~~\* set 11 p 11,~~

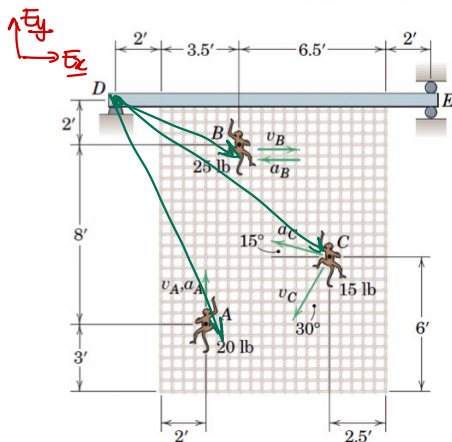
~~\* set 9 p 8~~

\* sets 14



w weight of bar and rope.

4/9 The monkeys of Prob. 4/8 are now climbing along the heavy rope wall suspended from the uniform beam. If monkeys A, B, and C have velocities of 5, 3, and 2 ft/sec, and accelerations of 1.5, 0.5, and 2 ft/sec<sup>2</sup>, respectively, determine the changes in the reactions at D and E caused by the motion and weight of the monkeys. The support at E makes contact with only one side of the beam at a time. Assume for this analysis that the rope wall remains rigid.



PROBLEM 4/9

$$\begin{aligned} a_B &= -0.5 \mathbf{e}_x \text{ ft/sec}^2 \\ a_A &= 1.5 \mathbf{e}_y \text{ ft/sec}^2 \\ a_C &= 2(-\cos 15^\circ \mathbf{e}_x + \sin 15^\circ \mathbf{e}_y) \text{ ft/sec}^2 \end{aligned}$$

BOLM  $F = mac$   
 $R_D + N_E + W = 0$

$$\begin{aligned} M^D &= (r_E - r_D) \times N_E \\ &+ (r_E - r_D) \times W \\ &= 0 \end{aligned}$$

BOLM

about D  
(fixed point)

$$\begin{aligned} M^D &= (r_E - r_D) \times (N_E + \Delta N_E) \\ &+ (r_B - r_D) \times W_B + (r_A - r_D) \times W_A + (r_C - r_D) \times W_C \\ &+ (r_E - r_D) \times W \end{aligned}$$

$$\begin{aligned} H^D &= \sum (r_i - r_D) \times m_i v_i \\ &= (r_A - r_D) \times m_A v_A + (r_B - r_D) \times m_B v_B + (r_C - r_D) \times m_C v_C \end{aligned}$$

$$\begin{aligned} \dot{H}^D &= \cancel{(r_A - r_D) \times m_A v_A} + (r_A - r_D) \times m_A a_A + (r_B - r_D) \times m_B a_B \\ &+ (r_C - r_D) \times m_C a_C \end{aligned}$$

$$M^D = \dot{H}^D \Rightarrow \text{solve for } \Delta N_E$$

$$\begin{aligned} R_D + \Delta R_D + N_E + \Delta N_E + W_A + W_B + W_C + W &= \sum m_i a_i = mac \\ &= m_A a_A + m_B a_B + m_C a_C \end{aligned}$$

- 3 scalar unknown
- 2 scalar equations

$$M^D = \dot{H}^D$$

$$\Delta N_E \mathbf{e}_y$$

8. [03-192] To avoid confusion label  $r$  in the figure  $R$  and the angle  $\theta$  requested in the solution as  $\beta$ .

Step 1. Choose the origin  $O$  to be at the bottom of the funnel and setup the cylindrical-polar coordinate system. Derive  $\mathbf{v}$  but not  $\mathbf{a}$ , we will not need it as we will solve the problem by exploiting conservations. The particle is constrained to move on a surface of revolution given by  $z^2 + (r - 1.15R)^2 = R^2$ . A time derivative of this expression yields  $z\dot{z} + \dot{r}(r - 1.15R) = 0$ .

Step 2. Draw a free-body diagram of the particle. Express the normal force as  $\mathbf{N} = N\mathbf{n}$ , where  $\mathbf{n}$  is a unit direction normal to the surface of revolution. In theory,  $\mathbf{n}$  could be computed from a gradient of  $z^2 + (r - 1.15R)^2 = R^2$ , but you don't need to do that here. You only need to note that  $\mathbf{N}$  has  $\mathbf{e}_r$  and  $\mathbf{e}_z$  components.

Step 3. In Step III, prove a conservation on the total mechanical energy  $E$  and a conservation of  $\mathbf{e}_z$  components of the angular momentum  $\mathbf{H}^O$ . You will need to refer to your FBD to identify these conserved quantities.

Step 4. Calculate the numerical values of  $E$  and  $\mathbf{H}^O \cdot \mathbf{e}_z$  using the initial conditions and complete your analysis.

$$1) z^2 + (r - 1.15R)^2 = R^2$$

$$z\dot{z} + (r - 1.15R)\dot{r} = 0$$

2)  $\mathbf{W} = -mg\mathbf{e}_z$   
 $\mathbf{N} = N\mathbf{n}$   
 $\mathbf{n} = n_r\mathbf{e}_r + n_z\mathbf{e}_z$   
 $\mathbf{N} \cdot \mathbf{v} = 0$

3) All the forces doing work on the system are conservative  $\Rightarrow$  the energy of the system is conserved.

$$T_B - T_A = W_{N,AB} = U_A - U_B$$

$$U = +mg\mathbf{e}_z \cdot \mathbf{r} = mgz$$

$$(C = -G \cdot \mathbf{r})$$

$$\frac{1}{2} m v_B^2 + mgz_B = \frac{1}{2} m v_A^2 + mgz_A$$

show that  $\mathbf{H}^O \cdot \mathbf{e}_z$  is conserved?

$$\mathbf{M}^O \cdot \mathbf{e}_z = 0 \Rightarrow \mathbf{H}^O \cdot \mathbf{e}_z \text{ is conserved}$$

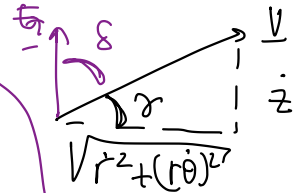
$$\mathbf{M}^O = \mathbf{r} \times \mathbf{f} = (r\mathbf{e}_r + z\mathbf{e}_z) \times (-mg\mathbf{e}_z + N(n_r\mathbf{e}_r + n_z\mathbf{e}_z))$$

$$= -rmgz(-\mathbf{e}_\theta) - rn_zN(\mathbf{e}_\theta) + zNr\mathbf{e}_\theta$$

$$\mathbf{M}^O \cdot \mathbf{e}_z = 0 \Rightarrow \mathbf{H}^O \cdot \mathbf{e}_z \text{ is conserved!}$$

We want to calculate  $U_B$ .

$$\mathbf{v} = \dot{r}\mathbf{e}_r + r\dot{\theta}\mathbf{e}_\theta + \dot{z}\mathbf{e}_z$$



$$\tan \gamma = \frac{\dot{z}}{\sqrt{\dot{r}^2 + (r\dot{\theta})^2}}$$

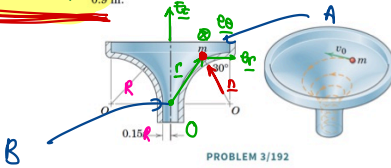
$$\mathbf{v} \cdot \mathbf{e}_z = \|\mathbf{v}\| \cos \delta$$

$$\frac{\pi}{2} - \delta = \gamma$$

$$\mathbf{r} = r\mathbf{e}_r + z\mathbf{e}_z$$

$$\dot{\mathbf{r}} = \dot{r}\mathbf{e}_r + r\dot{\theta}\mathbf{e}_\theta + \dot{z}\mathbf{e}_z$$

3/192 A particle is launched with a horizontal velocity  $v_0 = 0.55$  m/s from the  $30^\circ$  position shown and then slides without friction along the funnel-like surface. Determine the angle  $\beta$  which its velocity vector makes with the horizontal as the particle passes level  $O-O$ . The value of  $R$  is  $0.9$  m.



$$\mathbf{H}^O = \mathbf{r} \times m\mathbf{v}$$

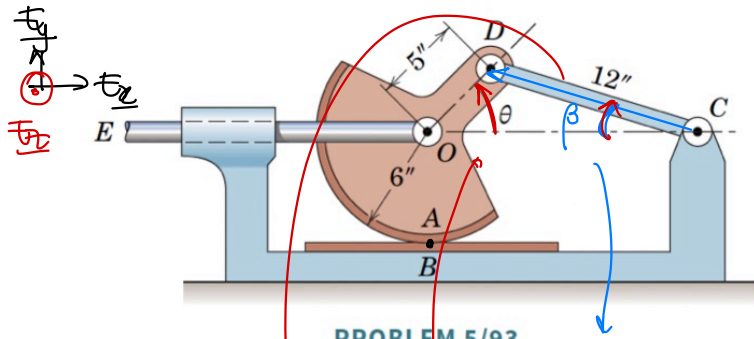
$$= m[(r\mathbf{e}_r + z\mathbf{e}_z) \times (r\dot{\theta}\mathbf{e}_\theta + \dot{z}\mathbf{e}_z)]$$

$$= m[r^2\dot{\theta}\mathbf{e}_z + r\dot{z}(-\mathbf{e}_\theta) + m\dot{r}\mathbf{e}_\theta - r\dot{z}\mathbf{e}_r]$$

$$\mathbf{H}^O \cdot \mathbf{e}_z = mr^2\dot{\theta}$$

$$mr_A^2 \dot{\theta}_A = mr_B^2 \dot{\theta}_B$$

5/93 A device which tests the resistance to wear of two materials A and B is shown. If the link EO has a velocity of 4 ft/sec to the right when  $\theta = 45^\circ$ , determine the rubbing velocity  $v_A$ .



PROBLEM 5/93

find  $v_A$   
 $v_o = 4 \text{ ft/sec}$

$$\frac{\sin \theta}{12} = \frac{\sin \beta}{5}$$

$$\omega_{OD} = \dot{\theta} \mathbf{e}_z$$

$$\omega_{DC} = \dot{\beta} (-\mathbf{e}_z)$$

$$\frac{\dot{\theta} \cos \theta}{12} = \frac{\dot{\beta} \cos \beta}{5} \Rightarrow$$

$$\underline{v_D} - \underline{v_O} = \underline{\omega_{OD}} \times (\underline{r_D} - \underline{r_O})$$

$$\underline{\omega_{OD}} = \omega_{OD} \mathbf{e}_z$$

$$\underline{OD} (\cos \theta \mathbf{e}_x + \sin \theta \mathbf{e}_y)$$

$$\underline{v_A} - \underline{v_O} = \underline{\omega_{OD}} \times (\underline{r_A} - \underline{r_O}) - \underline{OA} \mathbf{e}_y$$

$$\underline{v_A} = 4 \mathbf{e}_x + \omega_{OD} \mathbf{e}_z \times (-\underline{OA} \mathbf{e}_y)$$

$$= (4 + \omega_{OD} \underline{OA}) \mathbf{e}_x$$

$$\underline{v_D} - \underline{v_C} = \underline{\omega_{DC}} \times (\underline{r_D} - \underline{r_C})$$

$$\omega_{DC} \mathbf{e}_z$$

$$\underline{CD} (-\cos \beta \mathbf{e}_x + \sin \beta \mathbf{e}_y)$$

$$\underline{v_D} = \underline{v_O}$$

$$(1) \quad 4 \mathbf{e}_x + \omega_{OD} \mathbf{e}_z \times \underline{OA} (\cos \theta \mathbf{e}_x + \sin \theta \mathbf{e}_y) = \omega_{DC} \mathbf{e}_z \times \underline{CD} (-\cos \beta \mathbf{e}_x + \sin \beta \mathbf{e}_y)$$

$$\frac{\omega_{OD} \cos \theta}{12} = -\frac{\omega_{DC} \cos \beta}{5} \quad (2)$$

$$\dot{\theta} = \frac{\text{rad}}{\text{sec}}$$

$$\dot{\theta} = \frac{1 \text{ rev}}{2\pi \text{ sec}}$$

$$N = 2\pi \dot{\theta} = \frac{\text{rev}}{\text{sec}}$$

$$\dot{\theta} = \frac{N}{2\pi}$$

rad

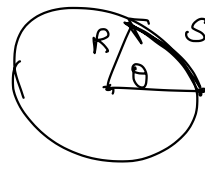
$$1 \text{ rev} = 2\pi \text{ rad}$$

$$\text{rad} = \frac{1 \text{ rev}}{2\pi}$$

$$N = \text{rev/sec}$$

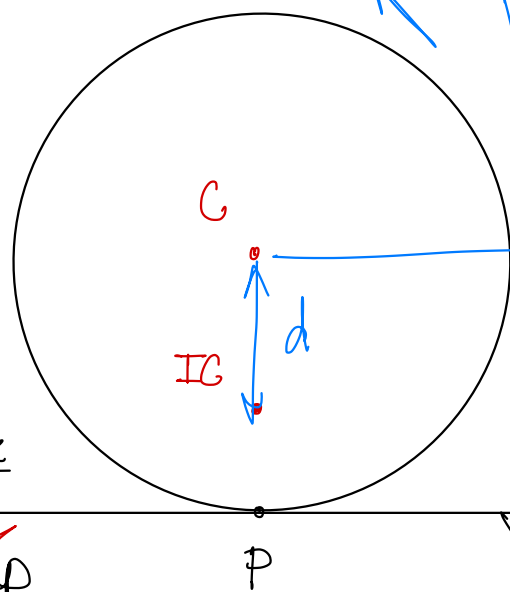
$$\frac{1 \text{ rev}}{\text{sec}} = \frac{2\pi \text{ rad}}{\text{sec}}$$

$$\theta = \frac{s}{R}$$



$$\underline{V_C} - \cancel{\underline{V_{IC}}} = \underline{\omega_{disk}} \times (\underline{r_C} - \underline{r_{IC}})$$

$$\underline{V_C} \underline{e_x} = \underline{\omega_{disk}} \underline{e_z} \times d \underline{e_y} = \underline{\omega_{disk}} d (-\underline{e_x})$$



$$\underline{V_C} = d \underline{\omega_{disk}}$$

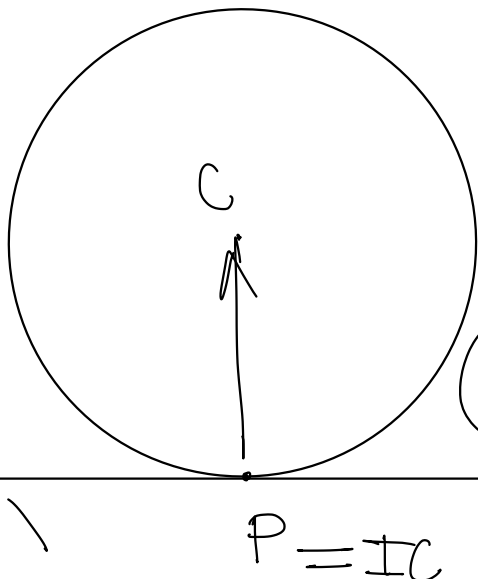
~~roll without slip~~  
 ~~$\underline{V_P} = \underline{0}$~~

roll with slip

$$\underline{V_P} \neq \underline{0}$$

$$\underline{V_P} \cdot \underline{e_y} = 0$$

$$1 \frac{\text{rev}}{\text{min}} = \frac{2\pi \text{ rad}}{60 \text{ s}}$$



$$\underline{V_C} - \cancel{\underline{V_P}} = \underline{\omega} \times (\underline{r_C} - \underline{r_P})$$

$$\underline{V_C} \underline{e_x} = \underline{\omega} \underline{e_z} \times R \underline{e_y} = -\underline{\omega} R \underline{e_x}$$

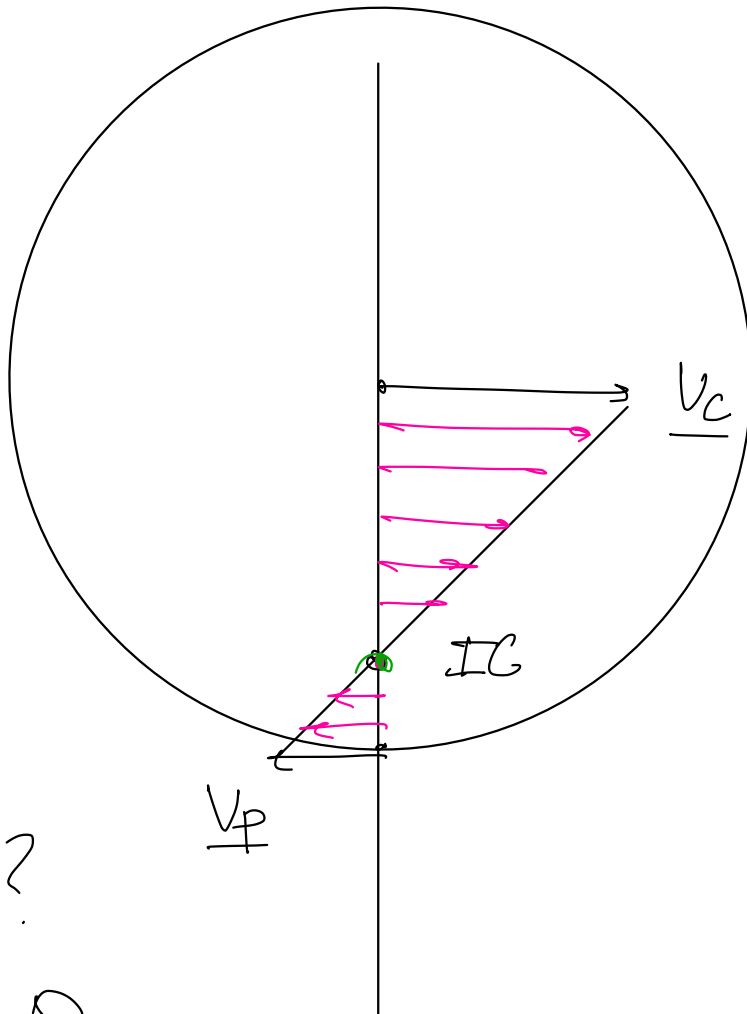
$$\underline{e_x} \cdot \underline{e_x} \quad \underline{V_C} = -\underline{\omega} R$$

$$a_C = -\alpha R$$

$$\underline{a_C} = -\alpha R \underline{e_x}$$

$$\underline{a_c} - \underline{a_p} = \underline{\alpha} \times (\underline{r_c} - \underline{r_p}) + \underline{\omega} \times (\underline{v_c} - \underline{v_p})$$

$$\underline{a_p} = \dots \underline{t_y}$$



$$\underline{r_{IC}} = ?$$

$$\underline{v_c} - \underline{v_{IC}} = \underline{\omega} \times (\underline{r_c} - \underline{r_{IC}})$$

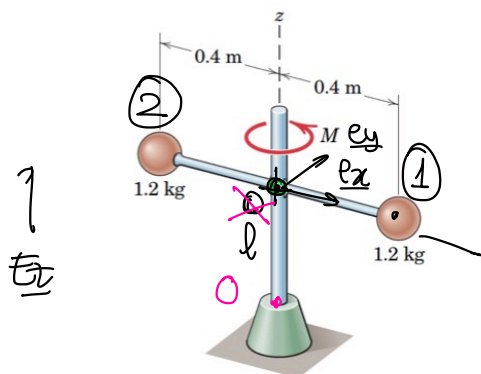
$$\underline{v_p} - \underline{v_{IC}} = \underline{\omega} \times (\underline{r_p} - \underline{r_{IC}})$$

2 equations, 2 unknowns.





3/178 The rigid assembly which consists of light rods and two 1.2-kg spheres rotates freely about a vertical axis. The assembly is initially at rest and then a constant couple  $M = 2 \text{ N} \cdot \text{m}$  is applied for 5 s. Determine the final angular velocity of the assembly. Treat the small spheres as particles.

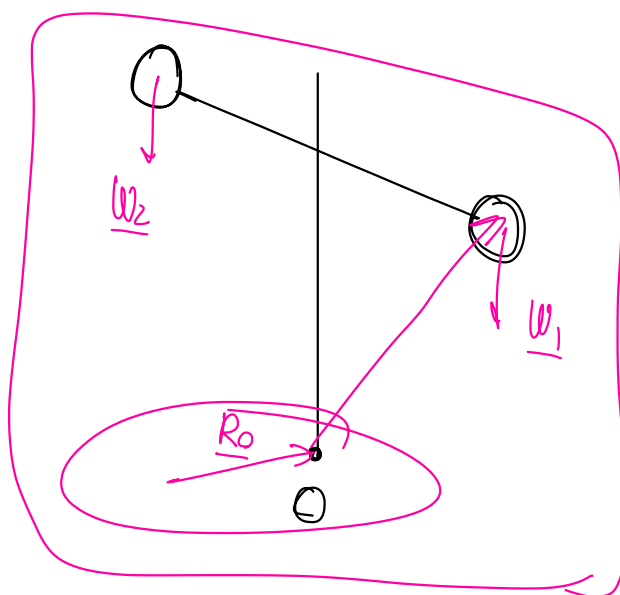


PROBLEM 3/178

$\underline{e}_z$

$$\underline{r} = 0.4 \underline{e}_z$$

$$\underline{v}_1 = 0.4 \dot{\theta} \underline{e}_\theta$$



$$\underline{r}_1 \times \underline{W}_1 + \underline{r}_2 \times \underline{W}_2 + \underline{M}$$

$$= (\underline{r}_1 \times \underline{W}_1 + \underline{r}_2 \times \underline{W}_2 + \underline{M})$$

$$= (\underline{r}_1 \times \underline{W}_1 + \underline{r}_2 \times \underline{W}_2 + \underline{M})$$

$$= \underline{M}$$

$$(2)(5) \underline{e}_z = (2)(0.4)^2 (m) \dot{\theta} \underline{e}_z$$

$\downarrow$   
at  $t=5$

$$\int_0^5 \underline{\dot{M}}^O dt = \underline{H}^O(t=5) - \underline{H}^O(t=0)$$

$$\underline{H}^O = (\underline{r}_1 + 0.4 \underline{e}_z) \times m (\dot{\theta} \underline{e}_\theta) + (\underline{r}_2 - 0.4 \underline{e}_z) \times m (-\dot{\theta} \underline{e}_\theta)$$

